

THE DESIGN OF ECONOMIC MICRO-IRRIGATION LATERALS BY COMPUTER AID

Abd El-Mageed, H. N.¹, M. M. Ibrahim², O. M. Elsherbiny³

ABSTRACT

To design micro-irrigation laterals, by an accurate, simple and quick method, The OPT-LAT model was created. The model calculates the maximum lateral length with field length in x-axis or y-axis at different lateral diameters. It can also estimate the optimal diameter that meets least cost irrigation system. Not only that, but also it can compare between optimal lateral diameter in x-axis or y-axis. The validity of OPT-LAT model was proved through the comparison between the lateral lengths calculated by the model and the corresponding ones calculated by Osama (2). The model has been applied on 160 m × 120 m plot area as a case study. At the x-axis direction, the lateral lengths were 58 m for different lateral diameters, the optimal internal diameter was 13.6 mm at annual least cost of 0.229 L.E/m/year and less percent of pressure head variation of 12 % at lateral diameter of 36 mm. The corresponding values at y-axis were 78 m lateral lengths, the optimal internal diameter was 13.6 mm at annual least cost of 0.239 L.E/m/year and less percent of pressure head variation of 12.04 % at lateral diameter of 36 mm. These results indicate that as the diameters of the laterals increased, the percent of the pressure head variation decreased, annual fixed cost increased and repair plus energy annual costs increased. The above results depend on energy price and pipe price according to its material type.

Keywords: *Computer model, Micro-irrigation lateral, Annual cost, Optimal diameter, Lateral length.*

¹ Prof. of Agric. Eng., Faculty of Agric., Al-Mansoura Univ.

² Assoc. Prof. of Agric. Eng., Faculty of Agric., Al-Mansoura Univ.

³ Demonstrator, Agric. Eng. Dept., Faculty of Agric., Al-Mansoura Univ.

INTRODUCTION

Micro-irrigation is one of the latest advances for applying water and it represents a definite progression in irrigation technology. The main problem in irrigation network planning is that there are no hard-fast scientific rules to depend on. Therefore, planning varies from one designer to another (**Ahmed, 1997**). A major challenge in system design is to select the optimum size and number of subunits that will achieve economical and efficient operation (**Keller and Karmeli, 1975**). Designing trickle irrigation systems is intensive in numerical calculations. The use of computers removes much of tedious work associated with repetitive complex calculations and the manipulation of large pools of data. This, results in less error frequency and more detailed analysis when compared to non-computer aided design (**Zazueta et al., 1985**). The most important advantages of the computer modeling, is the ability of analyzing interlaced or overlapping variables. In other words, if the change of one or more of the independent variables causes changing in other independent variables, and so on, this system, hence, is a complex system. Only computer modeling can trace these hundreds of operation and calculations till iteration steps in a specific condition. However, computer modeling is ideal for irrigation system analysis and design. The computer-aided design is necessary for accurate and quick design. So, bearing in mind the large number of alternatives under evaluation to select the optimal design, and the hardness of assessing each alternative. Hence, computer modeling must be used in this engineering process (**El-Nesr, 1999**). Few studies have reported the economic pipe size design. A computer model was used to assist the user in selecting the most economical (minimum total capital plus operational costs) design for trickle irrigation submain unit (**Sharaf, 1996**). A computer model called OSAMA (2) which calculates the maximum lengths of lateral and manifold according to the following limitation: (1) Water velocity in the lateral and the manifold should not exceed, 1.0 and 1.5 m/sec respectively, (2) Friction losses in the lateral and manifold should not exceed 55 % and 45 % of the total allowable losses, respectively, (3) Flow rate in the inlet of the unit should not exceed 12.5 and 25 m³/h

for single and double lateral unit, respectively. Program outputs were the maximum lateral length and dripper discharge at different flow variations (Suliman et al., 2004). Few studies have reported the performance characteristics of each emitter type. A computer model was developed for relating irrigation water temperatures to emitter discharge rate. According to this study, one may conclude that the pressure of the irrigation system is the dominant factor in the emitter discharge variation. Comes in descending order the emitter type, then the variation in irrigation water temperature (Ramadan et al., 2003). The manufacturing variation coefficient, emitter discharge coefficient and emitter discharge exponent were calculated due to establish flow sensitivity to pressure and compare manufacturers' specifications (Hezarjaribi et al., 2008). The results indicated that for the chosen emitters the manufacturers supplied data are not reliable for design purposes. Reliable, field tests are required prior to the design of a drip system. In fact, using the manufacturer's data will lead to non-uniformity of discharge throughout the system.

The main objectives of this study were to create a model able to estimate the most economic lateral design with respect to their number, maximum length and internal diameter in both directions.

METHODOLOGY

A model named OPT-LAT was created using Oracle Developer Suite 10g (forms) and Oracle Database 10g (PL/SQL Language) to achieve the main objectives of this study. OPT-LAT model database stored with information about the available diameter of Polyethylene pipe (PE) and number of brake horsepower hours per unit of fuel Table (1). The stored data can be inserted and edited or deleted at any time. OPT-LAT model was fed by emitter characteristics, type of connections, distance between emitters (S_e), design emission uniformity (EU), field length in x-axis (L_x), field length in y-axis (L_y) and input data for economical calculations. OPT-LAT model outputs were the maximum lateral length in x-axis (L_{Lx}) or y-axis (L_{Ly}) at different lateral diameters (D_i), number of lateral on both sides of manifold (N_{Lx}) or (N_{Ly}), percent of pressure head variation (H_{var}) and annual cost (TC_p).

Table (1): Typical number of brake horse power hours per unit fuel according to (USDA, 2013)

Fuel Type	Expected Brake Horsepower-hours
Diesel	4.0 hp.hr/liter
Natural gas	3.0 hp.hr/m ³
Electric	1.2 hp.hr/KWh at meter

The following steps were used to calculate the most economic pipe diameter for lateral:

The maximum allowable friction losses in the lateral (ΔH_f):

To calculate the maximum allowable friction losses in the lateral, the sequence of steps have been followed:

Firstly, emitter flow rates are characterized by empirically determining flow rates as a function of operating pressure. The flow and hydraulic pressure relations are given by (Pitts et al., 1986) as follows.

$$q_{av} = k_d \cdot h_{av}^x \dots\dots\dots (1)$$

Where:

h_{av} = The emitter operating pressure head, (m).

q_{av} = The emitter discharge, (L/hr).

K_d = The emitter discharge coefficient.

x = The emitter flow exponent.

Secondly, it can calculate minimum allowable emitter flow (h_n) according to (ASAE, 1990) who suggests the following equation (2) to estimate design emission uniformity in terms of C_v and pressure variation at the emitter, as well (Karmeli and Keller, 1974) cited by (Burt et al., 1997).

$$EU = 100 \left[1 - \frac{1.27 C_v}{\sqrt{N_e}} \right] \left(\frac{q_n}{q_{av}} \right) \dots\dots\dots (2)$$

$$q_n = \frac{EU \times q_{av}}{100 \left[1 - \frac{1.27 C_v}{\sqrt{N_e}} \right]} \dots\dots\dots (3)$$

$$\frac{q_{av}}{q_n} = \left(\frac{h_{av}}{h_n} \right)^x \dots\dots\dots (4)$$

Where:

q_n = The minimum emitter discharge in the subunit, (L/hr).

h_n = The minimum emitter operating pressure head in the subunit, (m).

EU = The design emission uniformity, (%).

C_v = The manufacturer's coefficient of variation.

N_e = The number of emitters per plant.

Thirdly, it can calculate the allowable pressure variation in the subunit (ΔH_s) that gives an EU reasonably close to the desired design value as to (Keller and Bliesner, 1990). ΔH_s is calculated according to the following equation:

$$\Delta H_s = 2.5 \times [h_{av} - h_n] \dots\dots\dots (5)$$

Where:

ΔH_s = The allowable subunit pressure head variation, (m).

Finally, the allowable friction losses in the lateral (ΔH_l) and manifold (ΔH_m) should not exceed 55% and 45 % of the allowable pressure variation in the subunit, respectively according to (Karmeli and Keller, 1975).

Maximum lateral length for both axes (L_{LX} , L_{LY}):

The program assumed that the number of laterals in both sides would be changed in ascending order starting from one until the total friction losses less or equal to the allowable friction losses (N_{LX} or $N_{LY} = 1, 2, 4, 6, 8,$ etc.). The length of one side lateral for both axes could be calculated as follows:

$$L_{LX} = \frac{L_X - s_p - (N_{LX} - 1) \cdot s_p}{2 \times N_{LX}} \dots\dots\dots (6)$$

$$L_{LY} = \frac{L_Y - s_p - (N_{LY} - 1) \cdot s_p}{2 \times N_{LY}} \dots\dots\dots (7)$$

Where:

L_{LX} = The length of on one side lateral in x-axis, (m).

L_{LY} = The length of on one side lateral in y-axis, (m).

L_X = The field length in x-direction, (m).

L_Y = The field length in y-direction, (m).

N_{LX} = The number of lateral along both sides manifold in x-axis.

N_{LY} = The number of lateral along both sides manifold in y-axis.

Total friction losses in the lateral (H_{fL}):

The following steps were used for calculating total friction losses:

a) The total number of emitters on lateral can be estimated as follows:

$$N_e = \frac{L_{LX}}{s_e} \dots\dots\dots (8)$$

$$N_e = \frac{L_{LY}}{s_e} \dots\dots\dots (9)$$

Where:

N_e = The total number of emitters along one side lateral.

s_e = The distance between emitters, (m).

b) The total discharge of one side lateral was calculated according the following equation:

$$q_L = N_e \cdot q_{av} \dots\dots\dots (10)$$

Where:

q_L = The total discharge of one side lateral, (L/hr).

c) The connection losses were calculated for on-line connection (sizes, large, medium and small connections), also for in-line connection. The equivalent length (f_e) is given by (**Montalvo, 1983**) and (**SCS, 1984**). The value of f_e is assumed as 0.23 for in-line connection and the f_e is calculated for on-line connection sizes according to the following equation:

$$f_e = \frac{23.04}{D_i^{1.84}} \text{ Large Connection } (D_e = 7.5 \text{ mm}) \dots\dots\dots (11)$$

$$f_e = \frac{18.91}{D_i^{1.87}} \text{ Standard connection } (D_e = 5 \text{ mm}) \dots\dots\dots (12)$$

$$f_e = \frac{14.38}{D_i^{1.89}} \text{ Small connection } (D_e = 3.8 \text{ mm}) \dots\dots\dots (13)$$

Where:

f_e = The equivalent length, (m).

D_i = The internal lateral diameter, (mm).

D_e = The on-line emitter protrusion diameter, (mm).

d) The friction head loss along the lateral line can be calculated by multiplying Eq. (15) by the Christiansen adjustment coefficient (F), according to (**Christiansen, 1942**). Reduction factor (F) depends on

the discharge exponent of friction loss formula and the number of outlets in the line. For multiple-outlet pipes, the total friction head loss is equal to the sum of the losses between the outlets. The following equation was used for calculating reduction factor:

$$F = \frac{1}{m+1} + \frac{1}{2N_e} + \frac{\sqrt{m-1}}{6 N_e^2} \dots\dots\dots (14)$$

Where:

F = The Christiansen adjustment coefficient (reduction factor).

m = The discharge exponent of friction loss formula = 1.852 for turbulent flow using the Hazen–Williams equation according to **(Wu et al., 1986)**.

Total friction losses in the lateral (H_{fL}) are determined using the Hazen-Williams equation:

$$H_{fL} = \frac{K \times L_L}{D_i^{4.87}} \times \left(\frac{q_L}{C}\right)^{1.852} \times F \times \left(\frac{S_e + f_e}{S_e}\right) \dots\dots\dots (15)$$

Where:

L_L = The length of one side lateral in x-axis L_{Lx} or in y-axis L_{Ly} , (m).

D_i = The internal lateral diameter, (mm).

K = The conversion constant, which is 1.212×10^{10} for metric unit

C = The friction coefficient, which is a function of pipe material, characteristics Hazen-Williams roughness coefficient for PE = 140 according to **(Keller and Bliesner, 1990)**.

The percent of pressure head variation (H_{var}):

For a fully laminar flow regime, emitters must be very sensitive to pressure head changes and the value of x must be 1.0. This means that a pressure variation of 20% may result in $\pm 20\%$ emitter flow rate variation. Most non-compensating emitters are always fully turbulent with an x level of about 0.5, indicating that a pressure variation of 20% will result in a flow variation of approximately 10% **(Solomon and Bezdek 1980; Boswell, 1985)**. The following steps were estimated the percent of pressure head variation:

$$h_{max} = h_{av} + 0.75 H_{fL} \dots\dots\dots (16)$$

$$H_{var} = \frac{h_{max} - h_n}{h_{max}} \dots\dots\dots (17)$$

Where:

H_{var} = The percent of pressure head variation, (%).

h_{max} = The allowable maximum emitter operating pressure head, (m).

The OPT-LAT model can calculate the percent of pressure variation (H_{var}) by gradually increasing to design emission uniformity (EU) when the percent of pressure variation was more than the percent of allowable emitter pressure variation.

Estimating the most economically lateral diameter:

The OPT-LAT model could be used to make comparison among different pipe diameters to estimate the most economically one. Input data for economic considerations were as follows: (1) pipe price according to its material type, (2) the number of years in the life cycle, (3) internal rate for capital, (4) type of engine used, (5) the equivalent annual rate of energy escalation, (6) brake horse power per unit of energy, (7) the annual numbers of hours to operate the pumps that was input data and (8) the following efficiencies were taken as 90%, 75% and 60% for electric engine, pump and internal combustion engine respectively. The sequence of steps to estimate the optimal lateral diameter have been followed:

- 1) The annual energy cost for lateral length can be estimated by using the following equation based on (SCS, 1984):

$$EC = \frac{(0.735) \times q_L \times H_{fl} \times T_{si} \times (C_e \times 1.36 / BHP) \times Ca}{75\eta} \dots\dots\dots (18)$$

Where:

EC = The annual energy cost of lateral, (L.E/year).

C_e = The fuel cost: diesel, (L.E/L); natural gas (L.E/m³); electricity, (L.E / KW.hr).

η = The overall pump efficiency, (%).

T_{si} = The annual numbers of hours to operate the pumps, (hr).

Ca = The equivalent annualized escalating energy cost factor.

BHP = The typical number of brake horsepower hours per unit fuel.

The equivalent annualized escalating energy cost factor, (Ca) is used to evaluate the effect of escalation on energy cost and can be calculated by (Jensen, 1981).

$$C_a = \left[\frac{(1+e)^N - (1+r)^N}{(1+e) - (1+r)} \right] \left[\frac{r}{(1+r)^N - 1} \right] \dots\dots\dots (19)$$

Where:

e = The decimal equivalent annual rate of energy escalation, (%)

r = The decimal equivalent annual interest rate, (%)

N = The number of years in the life cycle, (Years)

2) The annual fixed cost for lateral length has been calculated as follows:

$$FC = C_p \times CR \dots\dots\dots (20)$$

Where:

FC = The annual fixed cost of calculated lateral length, (L.E/year).

C_p = The pipe price that is a function of diameter and length, (L.E/m).

CR = The capital recovery factor can be calculated based on **(James and Lee, 1971)** as follows:

$$CR = \frac{r.(1+r)^N}{(1+r)^N - 1} \dots\dots\dots (21)$$

3) The annual maintenance cost (RC) varies from (1.5 to 2.5%) and (5 to 8%) of the initial cost for PE pipe and emitters respectively, according to **(Thompson et al., 1980)**.

4) Sum of the annual energy cost, the annual repair cost and the annual fixed cost of the pipe (TC_p) were estimated as follows:

$$TC_p = EC + FC + RC \dots\dots\dots (22)$$

Where:

TC_p = The annual cost, (L.E/year).

RC = The annual maintenance cost of the PE pipe, (L.E/year).

5) Finally, it can estimate annual cost of lateral for different diameters, (L.E/m/year) then select the optimal diameter that gives the least cost of system design.

OPT-LAT flowchart:

Steps for creating OPT-LAT are illustrated in Figure (1).

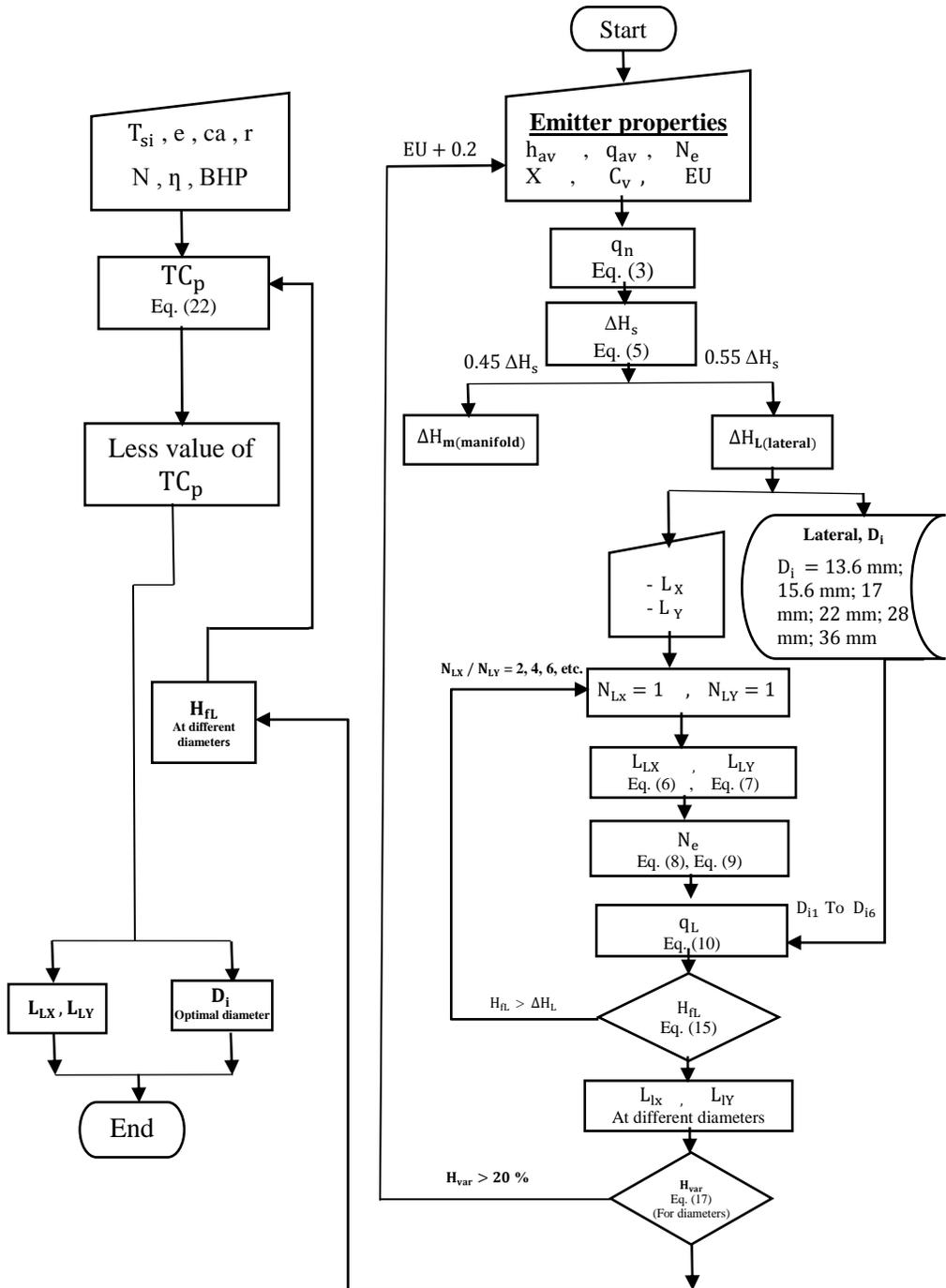


Figure (1): Flowchart describing the calculating economic pipe size for lateral.

RESULTS AND DISCUSSION

Model verification:

The verification of the OPT-LAT model was tested through the comparison with Osama (2) program. Table (2) indicates the values of input data for Osama (2) model. These inputs were considered and fed to design a model OPT-LAT. The emitter characteristics were presented in Table (2): the discharge rate for emitters were 2, 3, 4, 6 and 8 L/h and the connection type of emitters was on-line type. Other input, the flow variation was assumed 5% in this program. Internal lateral diameters of 13.6, 15.6 and 17 mm were stored in OPT-LAT model as a database. The design emission uniformity and emitter spacing were 91.5% and 0.5 m respectively. The output data of the two models were compared to the same entered input data as previously mentioned.

Table (2): Characteristics of the used emitters

q_{av} , (L/hr)	h_{av} , (m)	K_d	X	CV	D_e , (mm)
2	10	0.6325	0.5	0.030	3.8
3	11.763	0.6428	0.625	0.031	3.8
4	10	1.2649	0.5	0.027	5.0
6	11.073	2.3378	0.392	0.020	5.0
8	10	2.5298	0.5	0.030	5.0

q_{av} , h_{av} , K_d , X, CV and D_e are average emitter discharge (L/hr), emitter operating pressure head (m), proportion factor, discharge exponent, manufacturer's coefficient of variation and emitter protrusion diameter respectively.

It could be noticed that lateral lengths obtained from OPT-LAT model were close to the corresponding ones resulted by Osama (2) according to Table (3). These results prove the validity of OPT-LAT model in calculating the maximum allowable lateral length.

The regression coefficient (R^2) between calculated lengths of laterals from OPT-LAT and Osama (2) at different emitter discharge rates of 2, 3, 4, 6 and 8 L/hr are shown in Figure (2). The values of R^2 were 0.9996, 0.9991 and 0.9996 for internal lateral diameters of 13.6, 15.6 and 17 mm respectively. These values shows that the OPT-LAT model can calculate accurately lateral length.

Table (3): Lateral length at different emitter discharge rates and lateral diameters, (m)

D _i , (mm)	Lateral lengths, (m)	
	Osama (2) program output	OPT-LAT Predicted values
Emitter discharge rate = 2 L/hr		
13.6	72.5	75.5
15.6	92.5	95.5
17	114.5	113
Emitter discharge rate = 3 L/hr		
13.6	54	56.5
15.6	69	73
17	85.5	85.5
Emitter discharge rate = 4 L/hr		
13.6	45	47.5
15.6	58	60.5
17	72	71
Emitter discharge rate = 6 L/hr		
13.6	39.5	41
15.6	50.5	52.5
17	63	62
Emitter discharge rate = 8 L/hr		
13.6	29	30
15.6	37	39
17	46	45

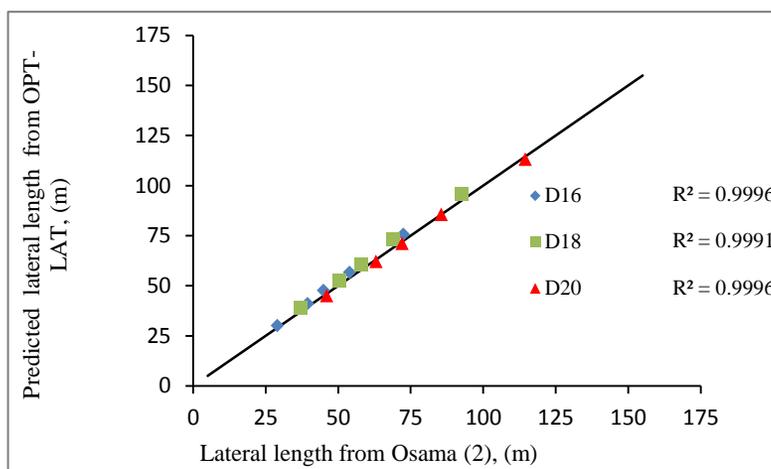


Figure (2): The correlation between lateral lengths from Osama (2) and predicted lateral lengths from OPT-LAT

Case study and analysis of the model results:

To carry out the case study, the sequence of input operations has been followed: Firstly, emitter characteristics, plant data, design area and soil data were presented in Table (4). Secondly, lateral diameters of 13.6, 15.6, 17, 22, 28 and 36 mm are shown in Figure (3). Thirdly, price of P.E lateral of available diameters and the price of on-line emitter are exhibited in Figure (4). Finally, economic considerations to lateral design are shown in Table (5) and interface for input economic variables and select type of engine are shown in Figure (5). It could select type of power source to diesel, electric and natural gas engine. From OPT-LAT model output results, the following studies were carried out; the effect of lateral diameters on (1) total friction losses in the lateral (2) allowable pressure head variation (3) annual fixed cost, sum of energy plus repair annual costs, and annual cost.

Table (4): Input data for the case study

Layout Parameters		
Input		
Plant	Plant name	Grape
	Tree spacing (m × m)	3 × 3
	Root depth (RZD), m	1.2
	Average daily peak (ETc), mm	4.1
Climate	Moderate	
Design area	Field length in X-direction (L_X), m	120
	Field length in Y-direction (L_Y), m	160
	Field Total Area (A), Fed	4.57
Soil	Soil type	Clay
	Hydraulic Conductivity coefficient (Ks), mm/hr	5
	Water-holding capacity of the soil (WHC), mm/m	192
Emitter	Average pressure head (h_{av}), Kpa	103
	Average emitter discharge (q_{av}), L/hr	4
	Discharge exponent (X)	0.45
	Discharge coefficient (K_d)	1.39
	Connection type	Standard
	coefficient of variation in the manufacturing (Cv), %	3.5
	Emission Uniformity (EU), %	90

Table (5): Input data for economic considerations

The service life for tube (N)	10 years
Internal rate for capital (r)	10%
Decimal equivalent annual rate of energy escalation (e)	15%
Type of engine used	Diesel
Brake horse power per unit of energy (BHP)	4 hp.hr/L
Unit price of diesel (C _e)	2.85 L.E/L
The annual numbers of hours to operate the pumps (T _{si})	300 hr
Pump efficiency (E _p)	75%
Efficiency of internal combustion engine (E _m)	60%

Figure (3): Interface for input available lateral diameters on the market

Figure (4): Interface for input price of P.E lateral per meter and price of emitter per piece

Figure (5): Interface for input economic variables and select type of engine

The main program interface for calculating the optimum lateral diameter is shown in Figure (6). The output of OPT-LAT model case study for

design economic diameter of lateral were shown in Table (6) and in addition that, minimum operating pressure head in the subunit, distance between emitter, emission points per plant and design emission uniformity were 90.7 Kpa, 1 m, 3 and 92 % respectively. The output data were as follows: maximum lateral lengths for both axes at different lateral diameters, number of laterals on both sides of the manifold at the two directions, percent of pressure head variation, annual fixed cost, repair plus energy annual costs and total annual cost of pipe.

Table (6): Output data for selection of economic lateral diameter

Lateral in X-axis										
L _x (m)	D _i (mm)	N _L	L _L (m)	q _L (L/hr)	H _{fL} (m)	ΔH _L (m)	H _{var} (%)	FC (L.E/m/Year)	RC + EC (L.E/m/Year)	TC (L.E/m/Year)
120	13.6	1	58	231.94	0.5755	1.73	15.45	0.1864	0.0419	0.229
	15.6	1	58	231.94	0.2867		13.74	0.219	0.0432	0.262
	17	1	58	231.94	0.1858		13.12	0.2515	0.0459	0.297
	22	1	58	231.94	0.0512		12.29	0.3004	0.0502	0.351
	28	1	58	231.94	0.0155		12.07	0.3329	0.0537	0.387
	36	1	58	231.94	0.0045		12	0.3654	0.0576	0.423
Lateral in Y-axis										
160	13.6	1	78	311.92	1.3316	1.73	19.62	0.1864	0.0526	0.239
	15.6	1	78	311.92	0.6633		15.95	0.219	0.048	0.267
	17	1	78	311.92	0.43		14.6	0.2515	0.049	0.301
	22	1	78	311.92	0.1184		12.71	0.3004	0.051	0.351
	28	1	78	311.92	0.0359		12.2	0.3329	0.054	0.387
	36	1	78	311.92	0.0104		12.04	0.3654	0.0576	0.423

The optimum lateral diameter - Lateral in X_Axis														
Do	Di	N	Lmax	Ne (L)	q	F	Fe	HfL (m)	Allow. HL (m)	Var H (%)	F.C, LE/Year/m	E.C, Repair, LE/Year/m	T.C, LE/Year/m	
<input type="checkbox"/>	16	13.6	1	58	58	231.942	.3593	.1435	.5755	1.728	15.45	.1864	.0429	.229
<input type="checkbox"/>	18	15.6	1	58	58	231.942	.3593	.1111	.2867	1.728	13.74	.219	.0432	.262
<input type="checkbox"/>	20	17	1	58	58	231.942	.3593	.0946	.1858	1.728	13.12	.2515	.0459	.297
<input type="checkbox"/>	25	22	1	58	58	231.942	.3593	.0584	.0512	1.728	12.29	.3004	.0502	.351
<input type="checkbox"/>	32	28	1	58	58	231.942	.3593	.0372	.0155	1.728	12.07	.3329	.0537	.387
<input type="checkbox"/>	40	36	1	58	58	231.942	.3593	.0232	.0045	1.728	12	.3654	.0576	.423

Figure (6): The main program interface for calculation the optimum lateral diameter

A rectangular plot area of 120 m × 160 m was chosen a case study to test the model. The length of 120 m was assumed as the x-axis of the area, while the length of 160 m was assumed as the y-axis of the area. At the x-axis direction, the lateral lengths were 58 m at lateral diameters. The

optimal inside diameter was 13.6 mm because it has the least annual cost which was 0.229 L.E/m/year. While the percent of pressure head variation with optimal diameter was 15.45 %, which is considered in the direction of x-axis. Furthermore, at y-axis direction, the lateral lengths were 78 m at lateral diameters. The optimal inside diameter was 13.6 mm because it has the least annual cost per meter which was 0.239 L.E/m/year. While the percent of pressure head variation with optimal diameter was 19.62 %, which is considered in the direction of y-axis. On the other hand, the least value of pressure head variation was 12 % at 36 mm lateral diameter at x-axis direction, but the least value of pressure head variation was 12.04 % at 36 mm lateral diameter at y-axis direction. Figure (7) shows a steep decline in total friction losses and percent of pressure head variation with the prime increase in lateral diameter (from 13.6 mm to 22 mm), after reaching to diameter 28 mm the declinations in H_f and H_{var} were very flat. The results of Figure (9) indicate that increasing internal diameter of the lateral makes approaching in the annual cost until 28 mm, after that increase in the annual cost due to increase lateral diameter over 28 mm is less than before. The previous interpretations are similar to Figure (10) and (12).

The correlation between internal lateral diameters, total friction losses and percent of pressure head variation are shown in Figure (7) and (10) for the x-axis and y-axis respectively. The regression coefficient (R^2) values between lateral diameters and total friction losses were 1 at the two directions. Moreover the values of R^2 between lateral diameters and percent of pressure head variation were approximately 0.895 and 0.903 for the x-axis and y-axis respectively. On the other hand, in Figure (8) and (11), the values of R^2 between internal lateral diameters and the sum of repair plus energy annual costs per meter were 0.994 and 0.795 for the x-axis and y-axis respectively. As well, the value of R^2 was 0.989 between internal lateral diameters and annual fixed cost per meter for both axes. Also, in Figure (9) and (12), the values of R^2 were 0.9906 and 0.9915 between internal lateral diameters and annual cost per meter for the x-axis and y-axis respectively.

The above results are very important when we get to choose the required diameter for the field. Increasing the diameter of laterals at the begin will

bring out significant effect on the depended variable such as friction losses, pressure head variation and annual cost while increasing the diameter over 28 mm has a slight effect on the annual cost.

Lateral in x-axis

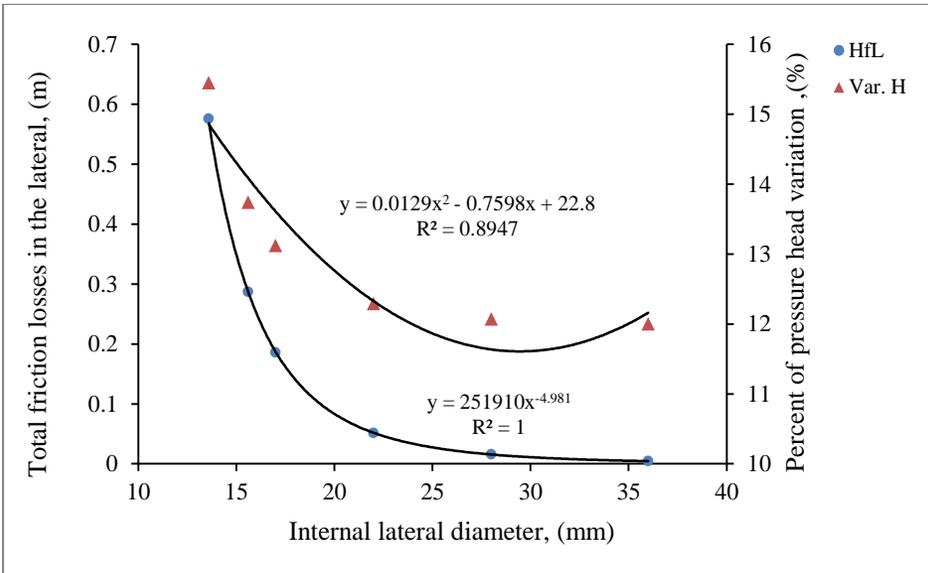


Figure (7): The correlation between internal lateral diameter, total friction losses and percent of pressure head variation at the x-axis.

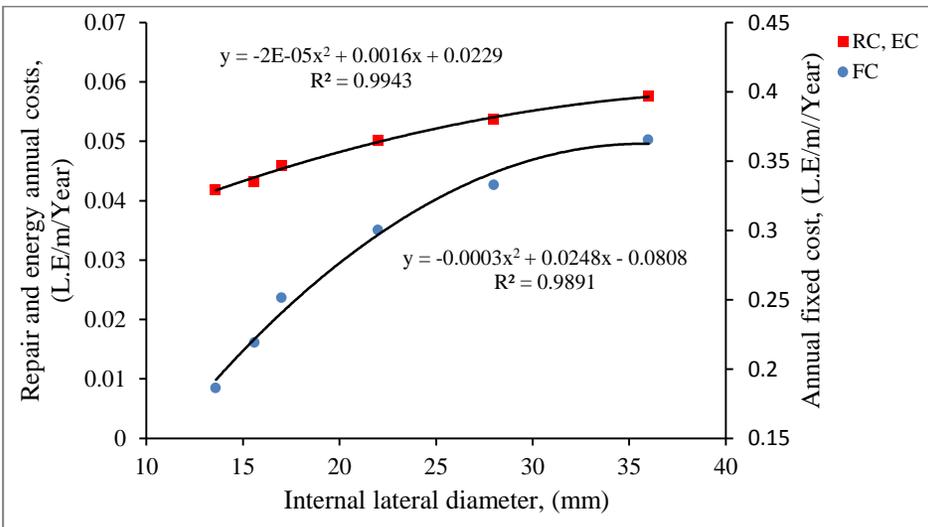


Figure (8): The correlation between internal lateral diameter, sum of repair plus energy annual costs per meter and annual fixed cost per meter at the x-axis.

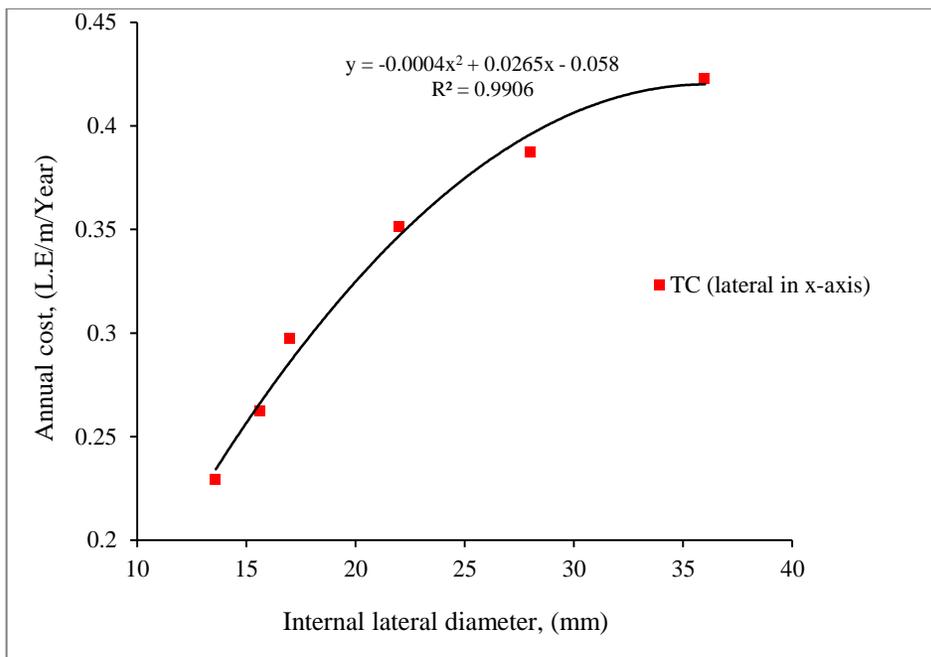


Figure (9): The correlation between internal lateral diameter and annual cost per meter at the x-axis.

Lateral in y-axis

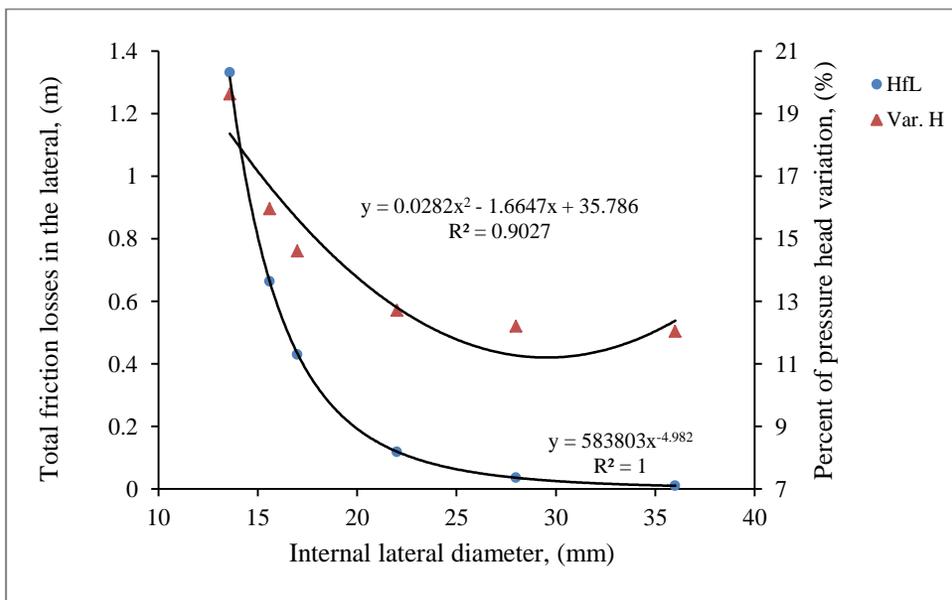


Figure (10): The correlation between internal lateral diameter, total friction losses and percent of pressure head variation at the y-axis.

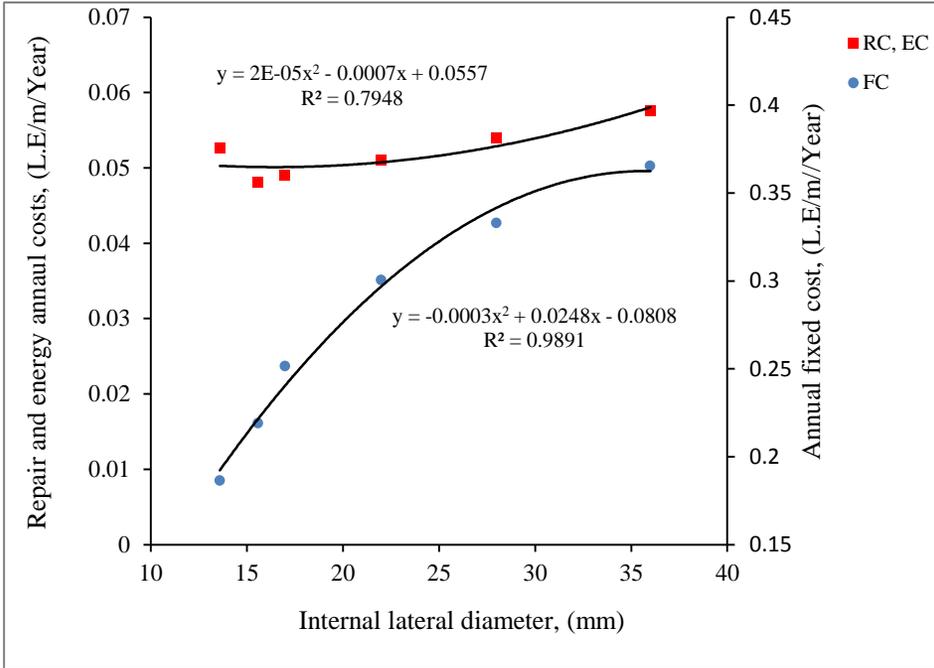


Figure (11): The correlation between internal lateral diameter, sum of repair plus energy annual costs per meter and annual fixed cost per meter at the y-axis.

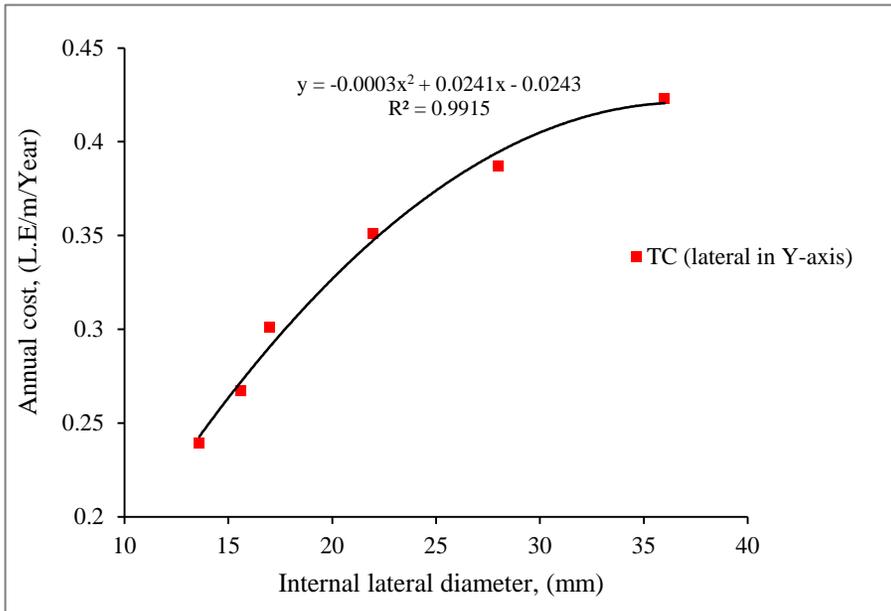


Figure (12): The correlation between internal lateral diameter and annual cost per meter at the y-axis.

CONCLUSIONS

The present study aimed to create a computer model called OPT-LAT to calculate optimize design of micro-irrigation laterals. Validity of OPT-LAT model was proved through comparison with Osama (2). The lateral lengths resulted from OPT-LAT model were close to the corresponding ones resulted by Osama (2). The regression coefficient values between predicted lateral lengths from OPT-LAT and lateral lengths from Osama (2) were 0.9996, 0.9991 and 0.9996 for lateral diameters 16, 18 and 20 mm respectively . This result proves the validity of OPT-LAT model in the calculation of lateral length for two directions. The OPT-LAT model is very good away for choosing the diameter of the lateral when the cost is taken in consider. From the management point of view, one can choose between lateral diameters without paying much and concern about the cost.

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الملخص العربي

التصميم الأقتصادي للخطوط الفرعية لأنظمة الري الميكرو بمساعدة الحاسب الآلي

أ.د/ هشام ناجي عبد المجيد^١ ، د/ محمد ماهر محمد إبراهيم^٢ و م/ أسامة محمد الشربيني^٣

تم اعداد برنامج (OPT-LAT) بغرض التصميم الأقتصادي للخطوط الفرعية لأنظمة الري الميكرو بطريقة سريعة ودقيقة. يمر البرنامج بعدة مراحل لتحديد أقصى طول للخط الفرعي وذلك مع القطر الذي يحقق أقل تكلفة اجمالية سنوية على النحو التالي: (١) عمل قاعدة بيانات لأقطار البولي إيثيلين المتاحة في الأسواق ويمكن التعديل أو الحذف بها في أى وقت (٢) تغذية البرنامج بالمدخلات وهي عبارة عن خصائص النقاط - المسافة بين النقاطات - انتظامية التوزيع للنقاطات المراد التصميم عليها (٣) تكلفة الخط الفرعي بالجنيه لكل متر (٤) طول وعرض مساحة التصميم (٥) الاعتبارات الأقتصادية (٦) القدرة الحصانية لمصدر الطاقة المستخدمة سواء كان سولار أو غاز طبيعي أو كهرباء . ثم يقوم البرنامج بحساب أقصى طول للخط الفرعي وحساب عدد الخطوط الفرعية الموضوعه على المشعبات وذلك في الاتجاهين (X,Y) للأرض بحيث لا يتعدى الفواقد الناتجة عن الاحتكاك الفواقد المسموح بها. ثم تتم المقارنة الأقتصادية بين الأقطار الداخلية المختلفة عن طريق حساب التكاليف الاجمالية السنويه للمتر الواحد لكلا من التكاليف الثابتة وتكاليف الطاقة وتكاليف الصيانة. طول الخط الفرعي الأمثل عندئذ يكون عند القطر الداخلى الذي يحقق أقل تكاليف إجمالية سنوية للمتر الواحد.

إثبات صحة البرنامج:

لتجربة البرنامج تم تقييم المخرجات المتحصل عليها من برنامج (OPT-LAT) مع برنامج آخر (Osama (2)) وكانت الأطوال المحسوبة للخط الفرعي متقاربة جدا ويعمل علاقة بين الأطوال المحسوبة كانت قيم معامل الارتباط للأطوال عند الأقطار الداخلية المختلفة (٦.١٣ - ١٥.٦ - ١٧مم) تساوى (٠.٩٩٩٦-٠.٩٩٩١-٠.٩٩٩٦) وهذه القيم توضح دقة وصحة البرنامج في حساب طول الخط الفرعي.

^١ أستاذ بقسم الهندسة الزراعية - كلية الزراعة - جامعة المنصورة

^٢ أستاذ مساعد بقسم الهندسة الزراعية - كلية الزراعة - جامعة المنصورة

^٣ معيد بقسم الهندسة الزراعية - كلية الزراعة - جامعة المنصورة

دراسة حالة:

تم عمل دراسة حالة للبرنامج لمساحة أرض ابعادها فى اتجاه (X) تساوى ١٢٠ متر وفى اتجاه (Y) تساوى ١٦٠ متر والمقارنة بين الأقطار الداخلية المختلفة (١٣.٦-١٥.٦-١٧-٢٢-٢٨-٣٦م) وكانت البيانات المتحصل عليها من دراسة الحالة كالتالى:

١- عند تصميم الخط الفرعى فى اتجاه (X) : كان طول الخط الفرعى لمختلف الأقطار الداخلية ٥٨ متر والقطر الأمثل (الأقتصادى) ١٣.٦ مم عند أقل تكاليف إجمالية سنوية ٠.٢٢٩ جنيه /متر/سنه فى حين أن النسبة المئوية للاختلاف فى الضغط عند هذا القطر ١٥.٤٥% وأقل نسبة مئوية للاختلاف فى الضغط كان عند القطر الأكبر ٣٦ ملم الذى يساوى ١٢% .

٢- عند تصميم الخط الفرعى فى اتجاه (Y) : كان طول الخط الفرعى لمختلف الأقطار الداخلية ٧٨ متر والقطر الأمثل (الأقتصادى) ١٣.٦ مم عند أقل تكاليف إجمالية سنوية ٠.٢٣٩ جنيه /متر/سنه فى حين أن النسبة المئوية للاختلاف فى الضغط عند هذا القطر ١٩.٦٢% وأقل نسبة مئوية للاختلاف فى الضغط كان عند القطر الأكبر ٣٦ مم والذى يساوى ١٢.٠٤% .

٣- أوضحت النتائج المتحصل عليها من النموذج الرياضى إنه كلما زاد قطر الخط الفرعى يقل الفوائد الناتجة عن الاحتكاك وتقل النسبة المئوية للاختلاف فى الضغط ويزداد التكاليف الإجمالية السنوية للطاقة والصيانة وكذلك يزداد التكاليف الإجمالية الثابتة السنوية ولكن بمعدلات مختلفة تتوقف على سعر الطاقة المستخدمة وسعر الخط الفرعى وتتم المقارنة بين هذين الأمرين واختيار الاتجاه الأفضل (الأقتصادى) وكذلك المفاضلة بين اختيار القطر الأكبر أم اختيار القطر الأصغر أم التصميم فى اتجاه (X) أو (Y).